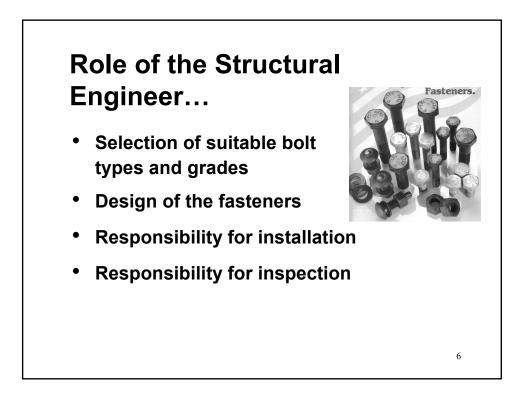
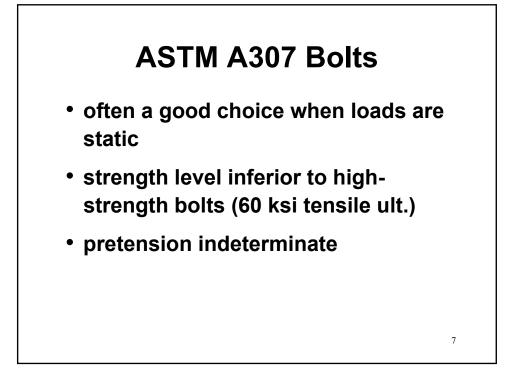


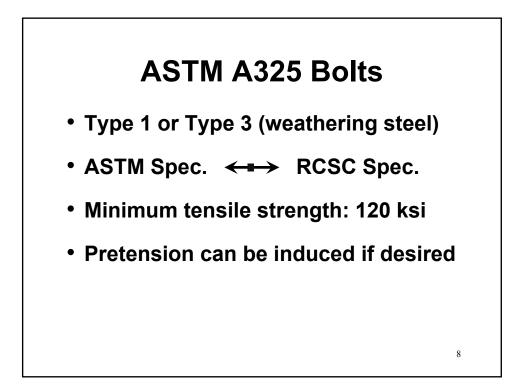
## **High-Strength Bolts: The Basics**

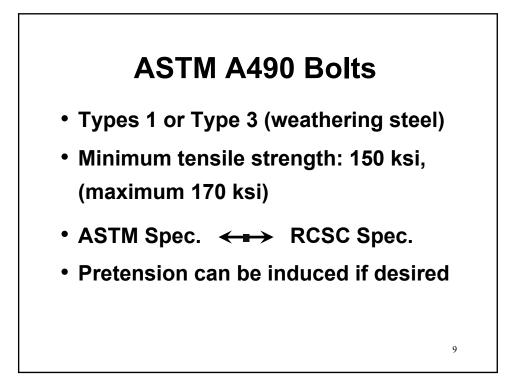
- Fundamentals and Behavior
- Specification Requirements (AISC 2010)

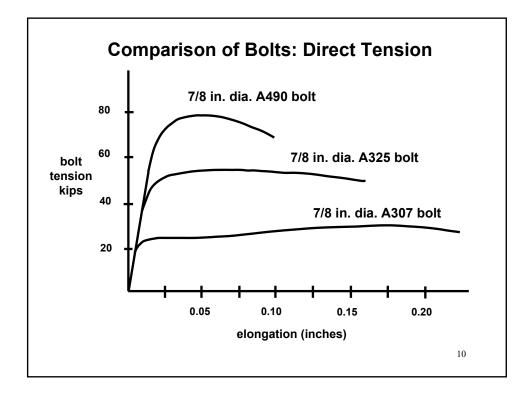


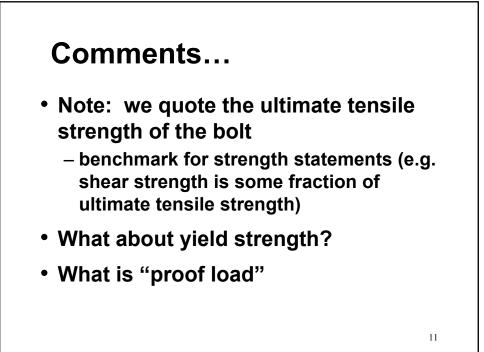


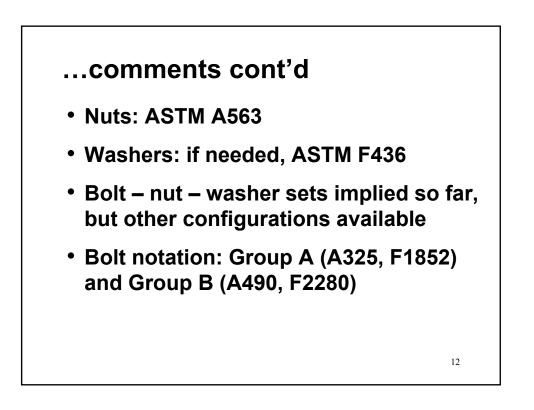


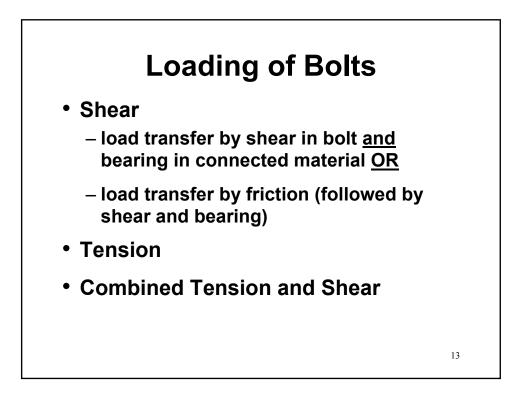


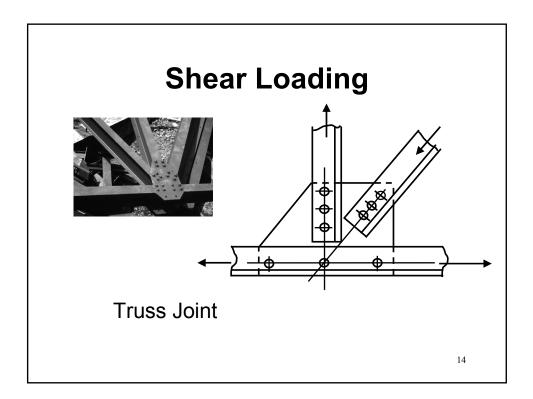


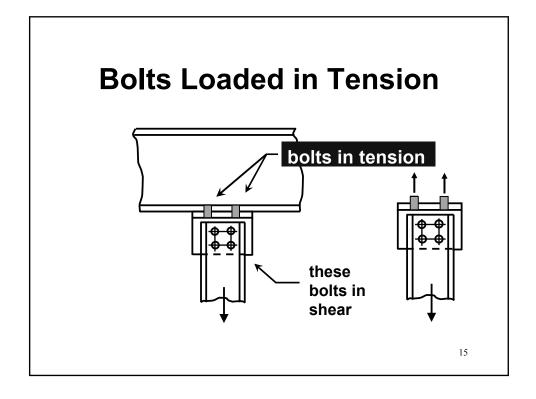


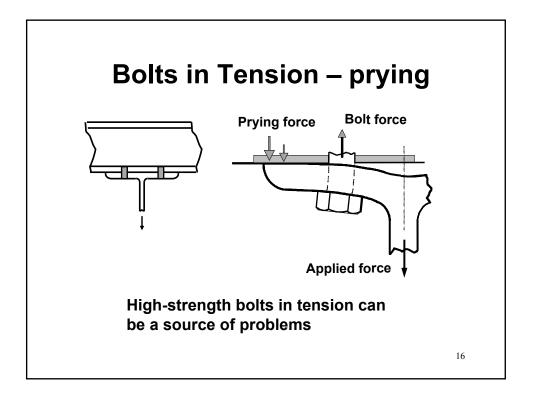


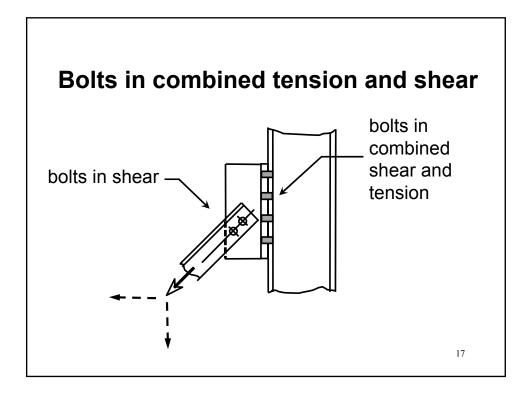


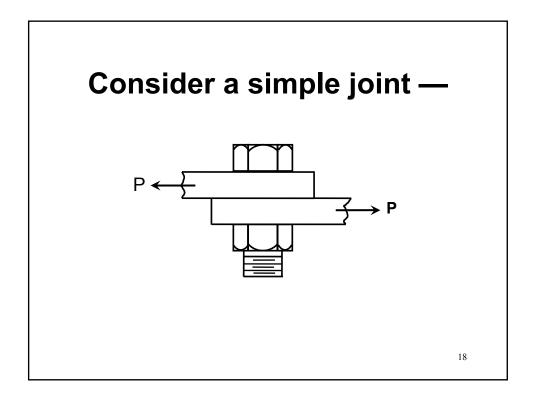


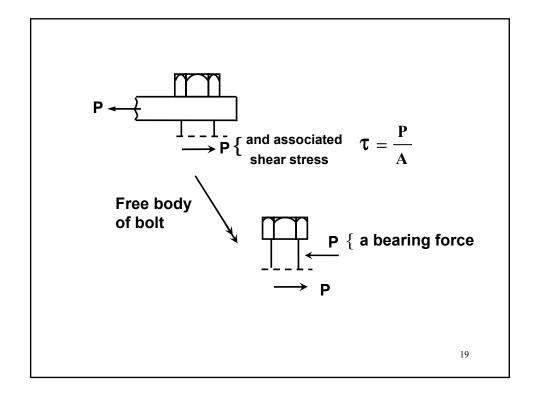


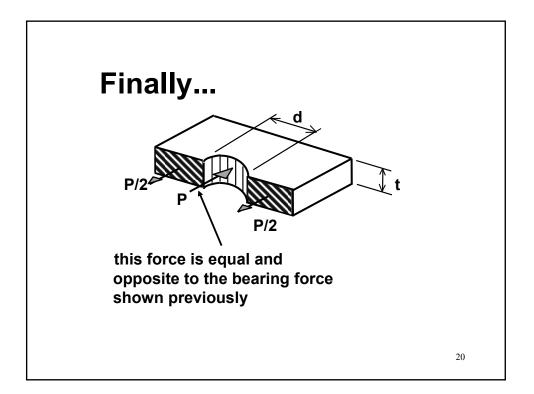


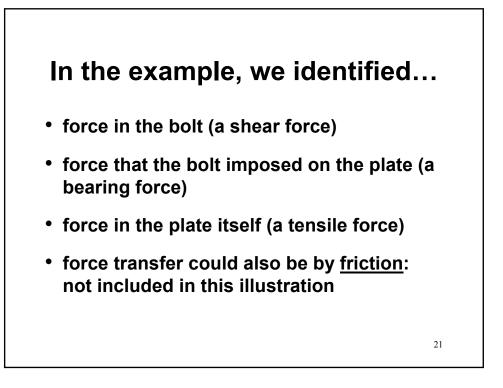


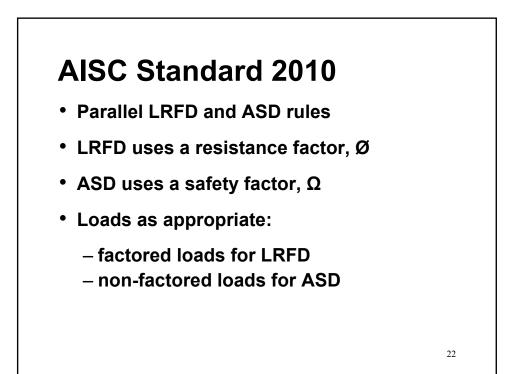


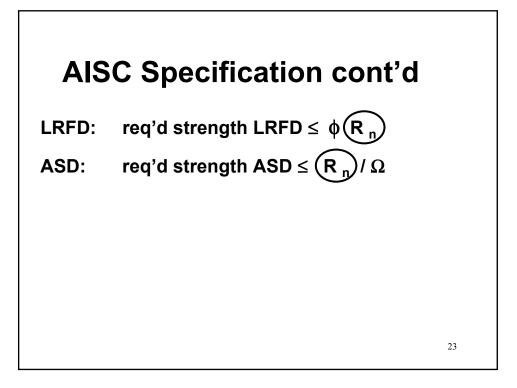


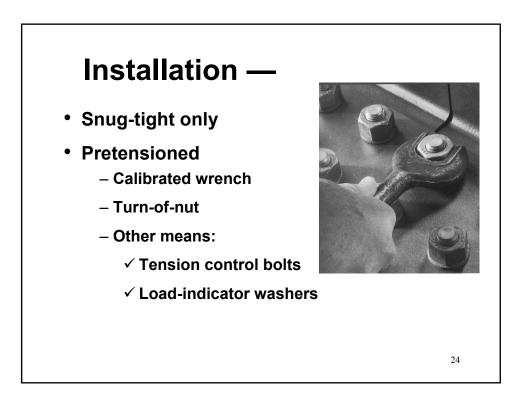


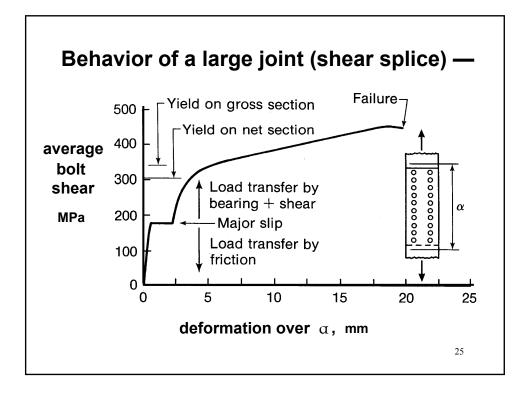


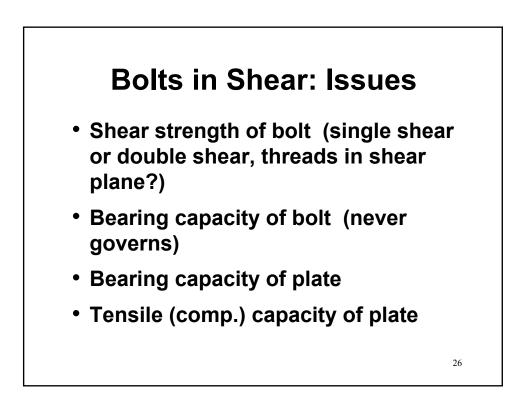


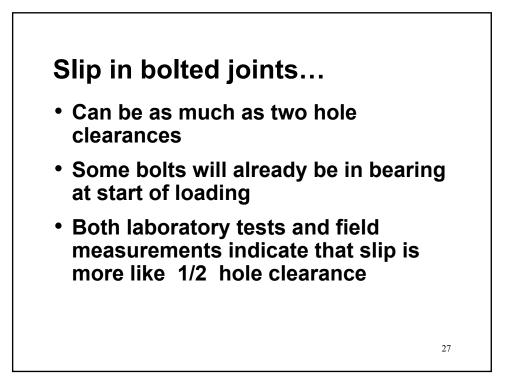


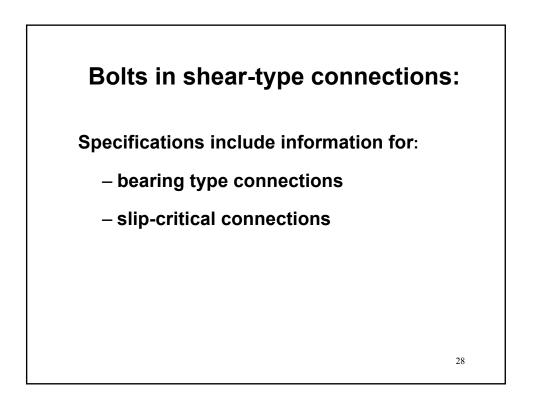


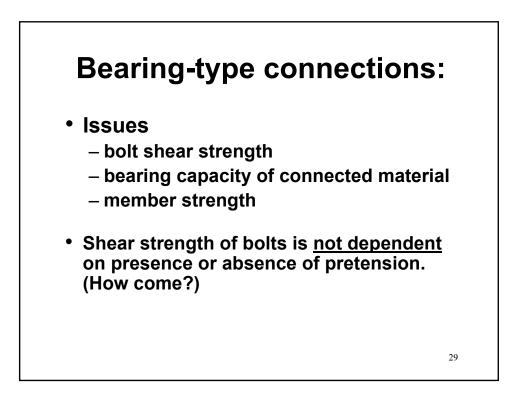


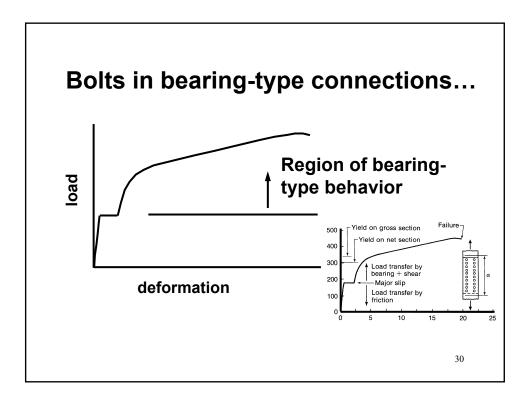


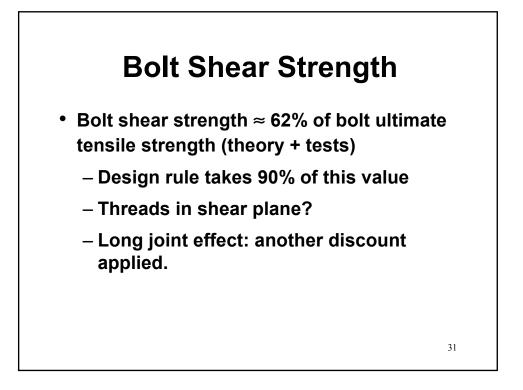


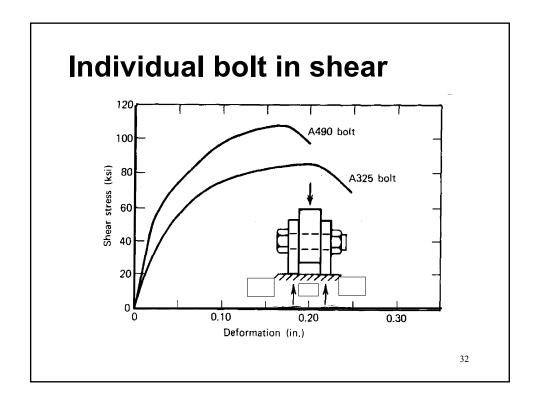


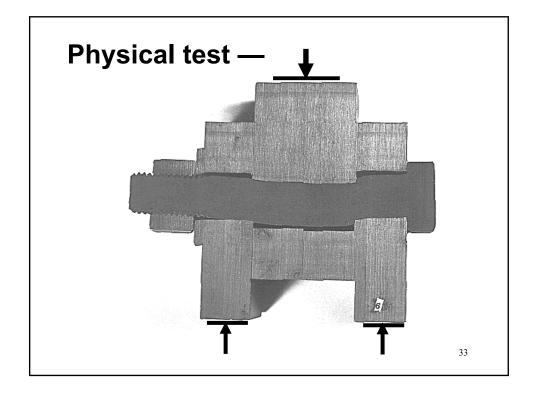


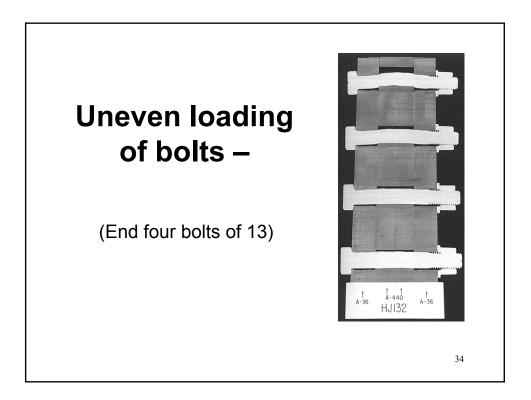


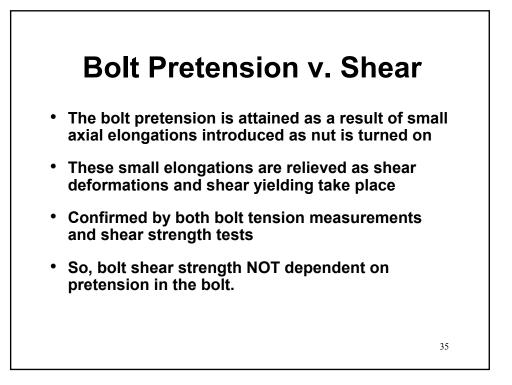


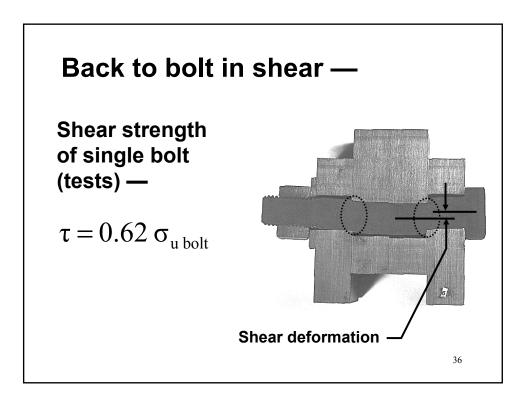


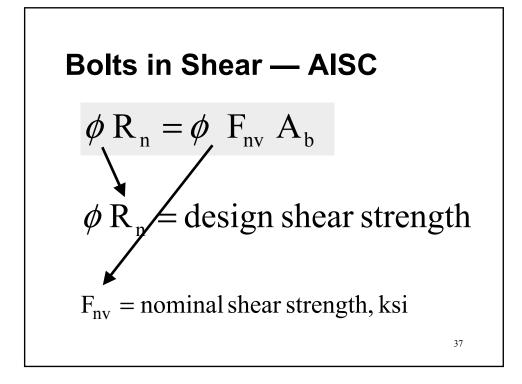


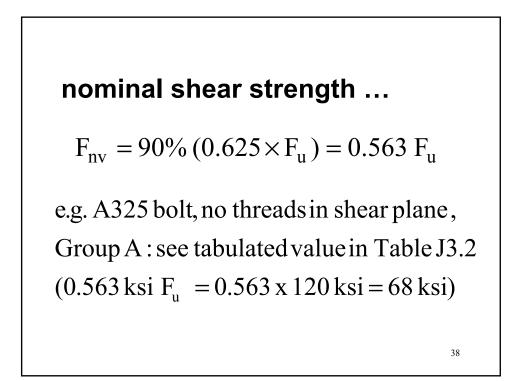












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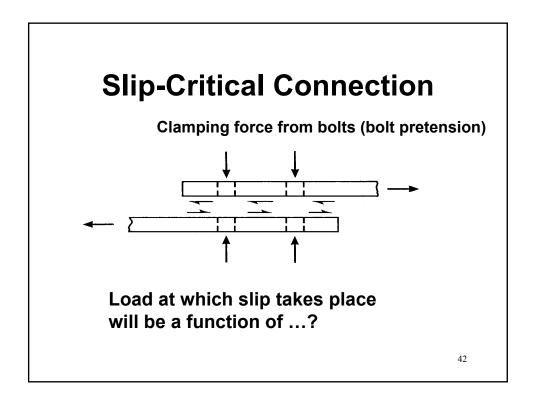
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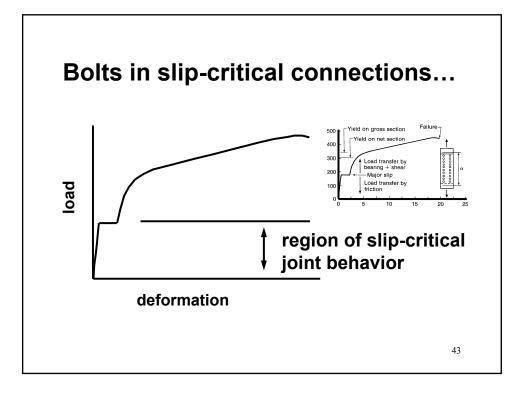
For threads <u>included</u>, the tabulated values are 80% of the above.

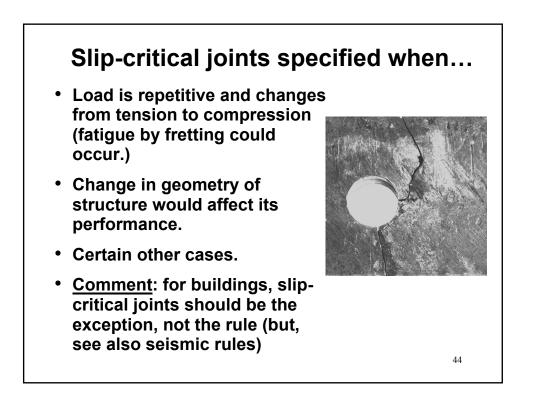
## Comments...

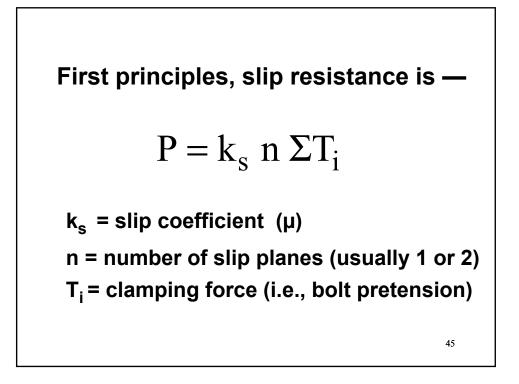
- The discount for length (use of 90%) is conservative
- If joint length > 38 in., a <u>further</u> reduction, to 83%
- The Ø value used for this case (0.75) is conservative

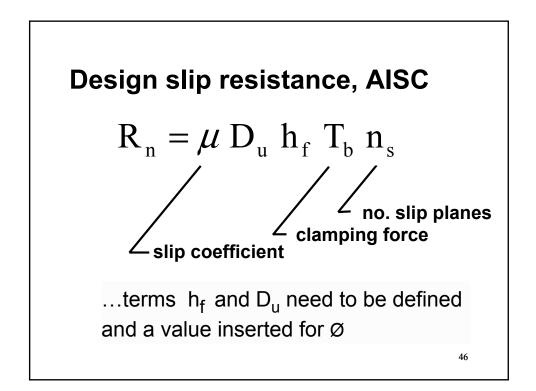








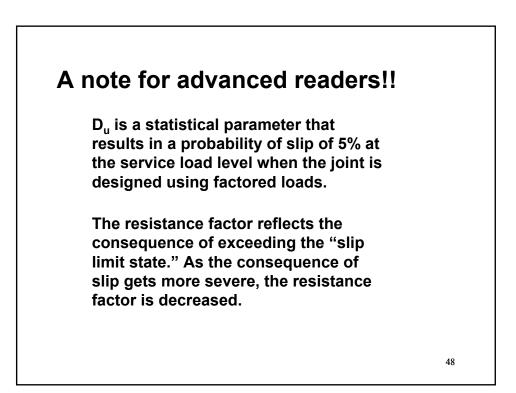


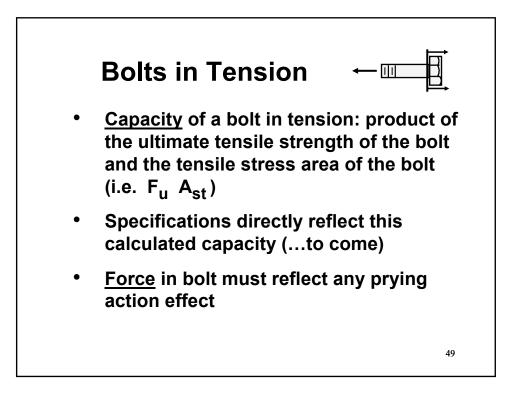


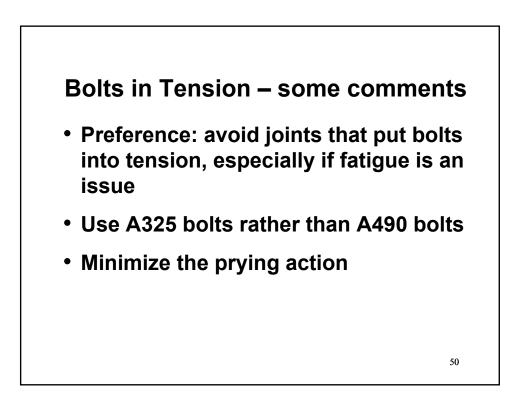
47

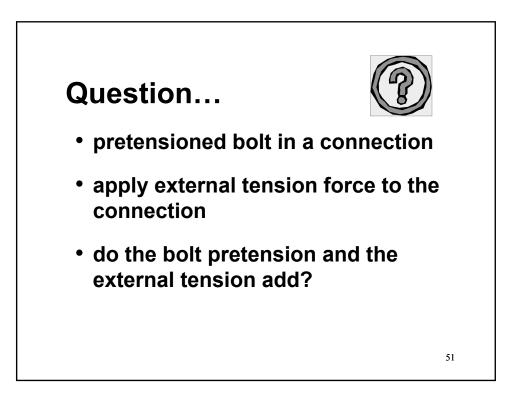
## and the modifiers ...

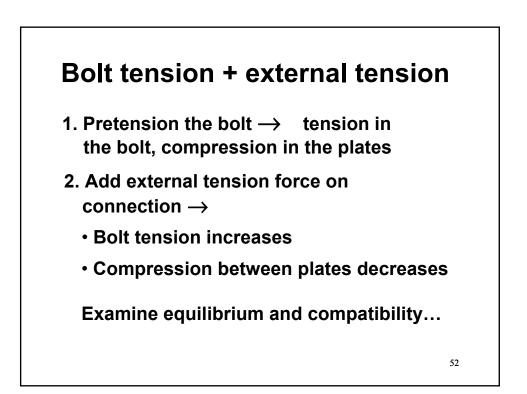
 $h_{f} = \text{modifier re fills : either 1.0 or 0.85}$   $\phi = 1.0 \text{ for std. holes and for short slots } \perp$  = 0.85 for oversize and short slots parallel = 0.70 for long slotted holes  $D_{u} = 1.13, \text{ ratio of installed bolt}$ tension to specified minimum bolt tension  $\mu = 0.30 \text{ clean mill scale, hot - dipped galvanized}$ and roughened, etc. (Class A surfaces)  $\mu = 0.50 \text{ unpainted and blast - cleaned, etc.}$ (Class B surfaces)

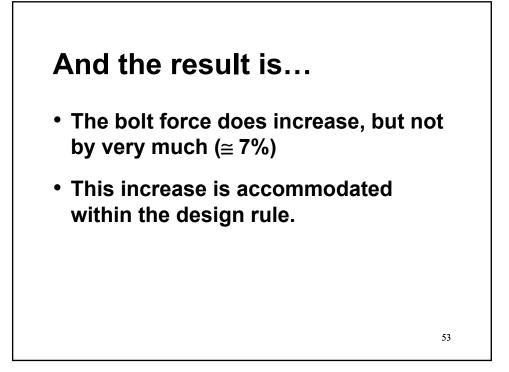


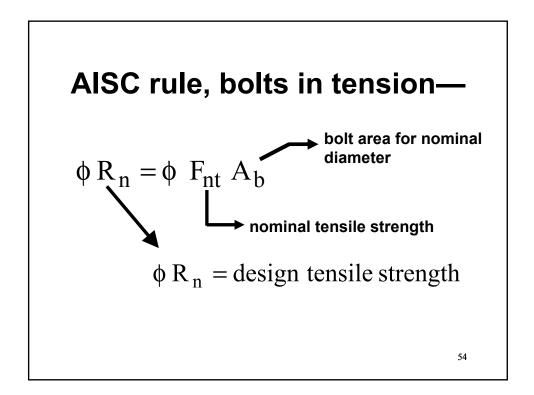


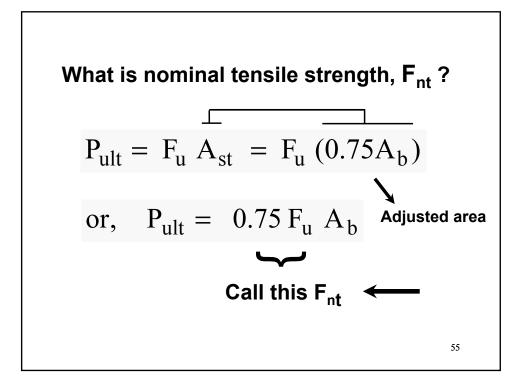




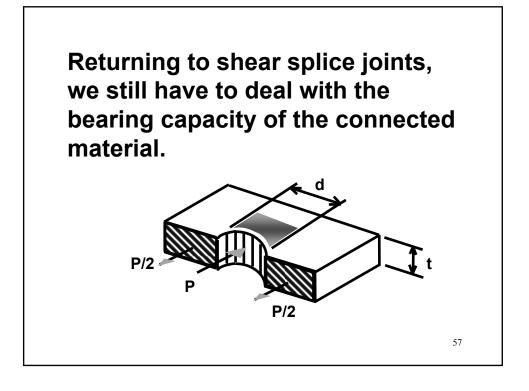


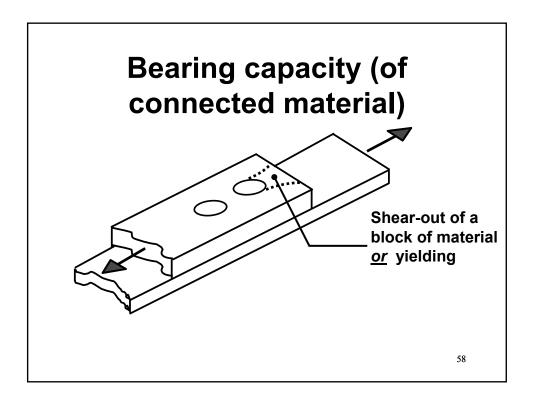


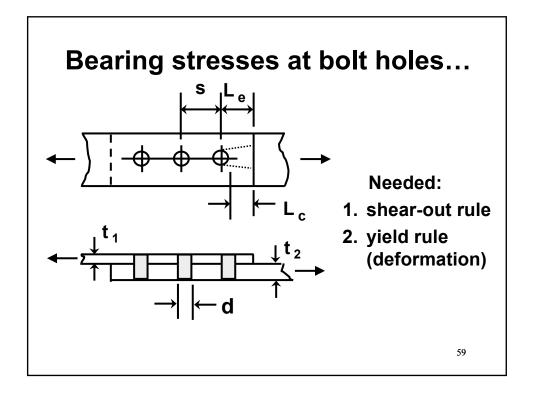


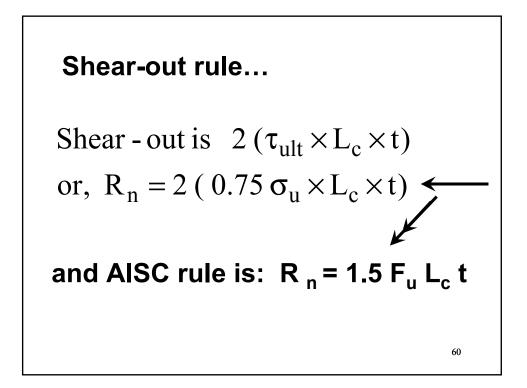


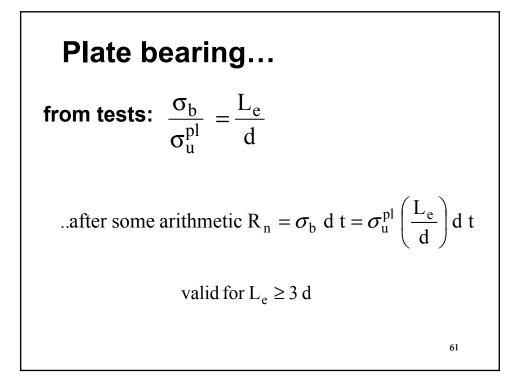
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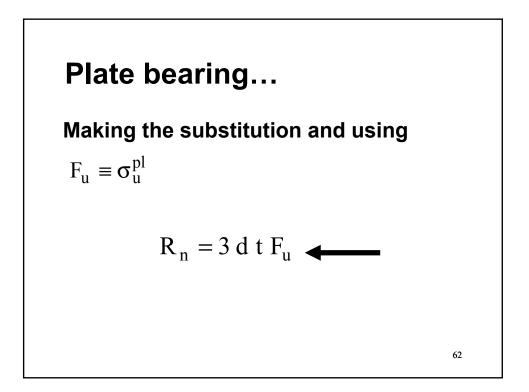












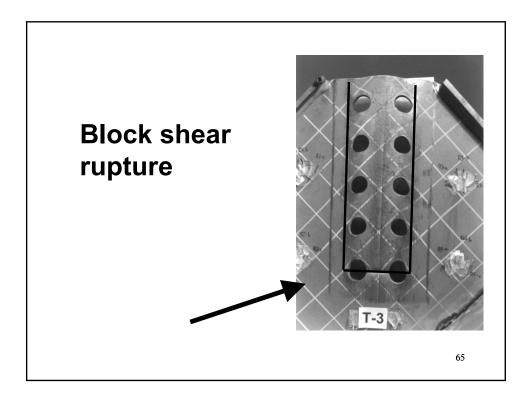
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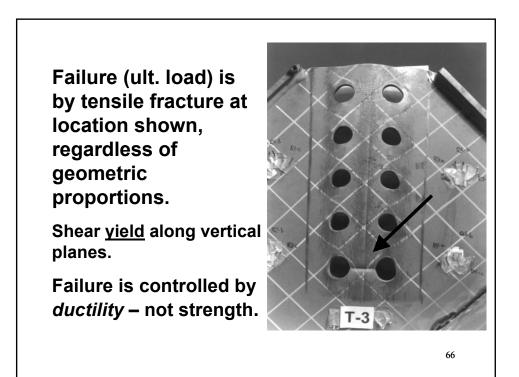
## Finally, the AISC rule for plate bearing capacity is ...

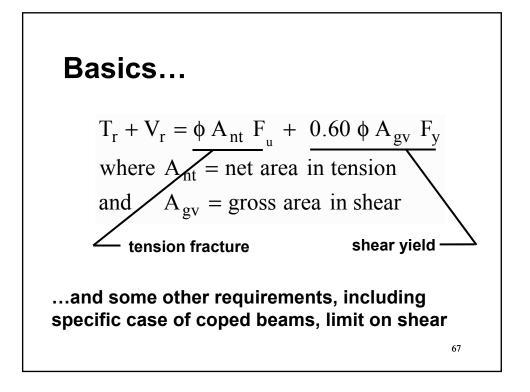
$$R_n = 1.5 F_u L_c t \le 3.0 d t F_u$$

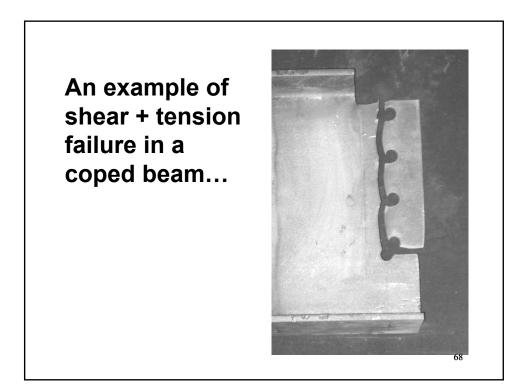
(with a  $\phi$ -value still to be inserted)

Further note re bearing... When deformation a consideration, use  $R_n = 1.2 F_u L_c t \le 2.4 d t F_u$ Why this difference, and when do we use the latter? (value of  $\phi$  still to be applied)

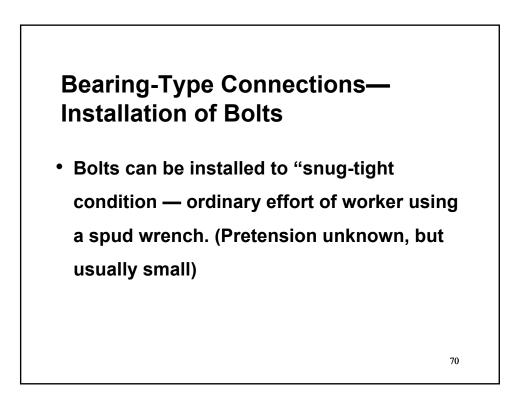


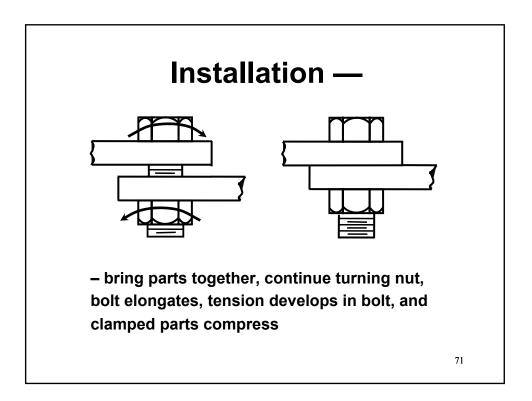


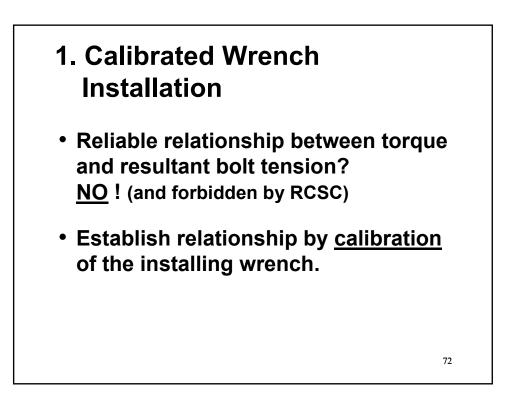


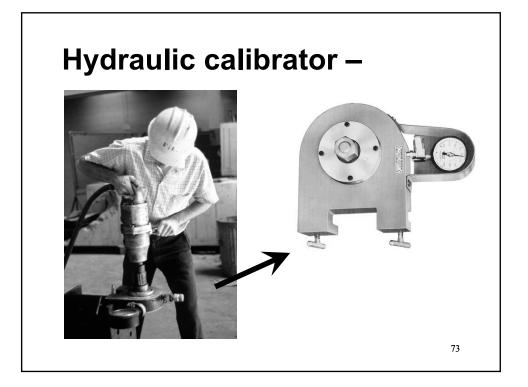


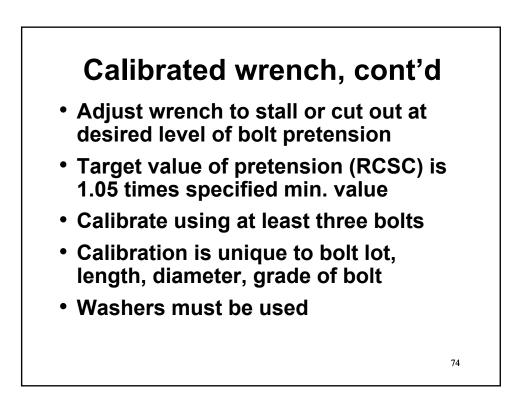


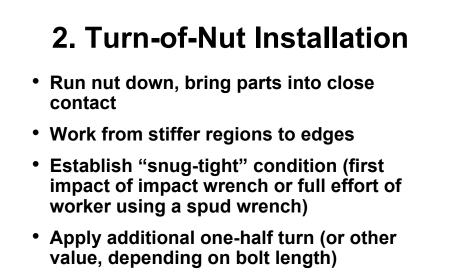


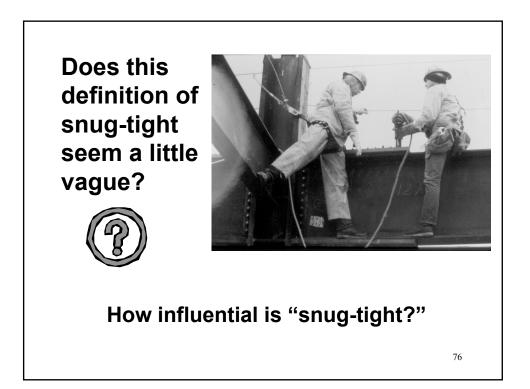


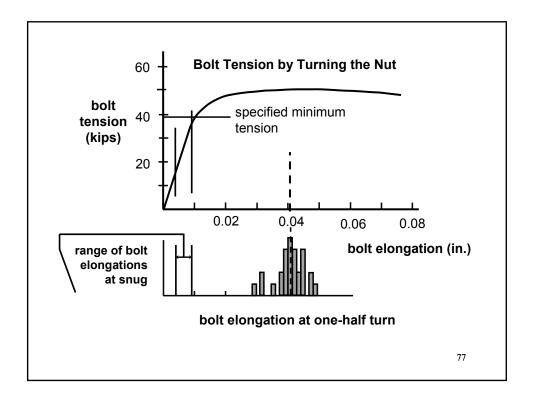


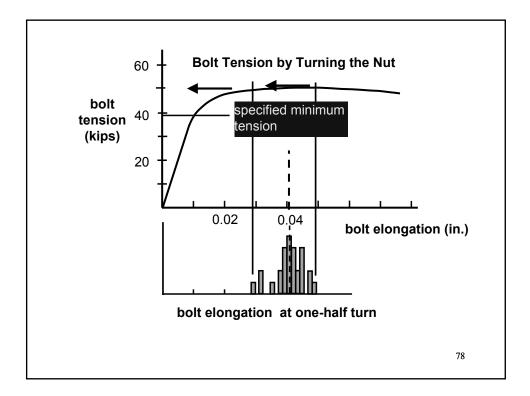


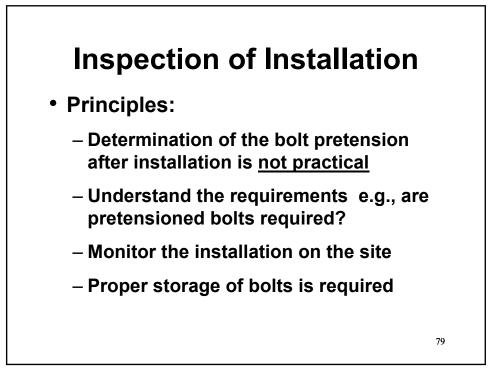


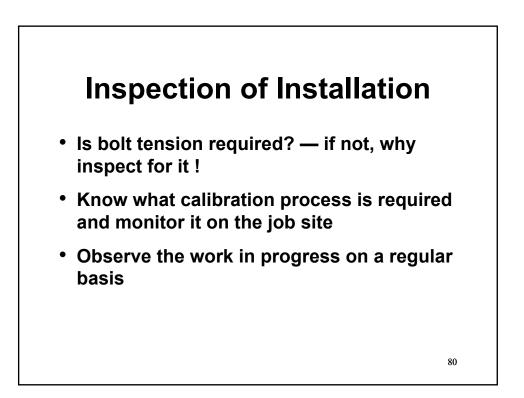


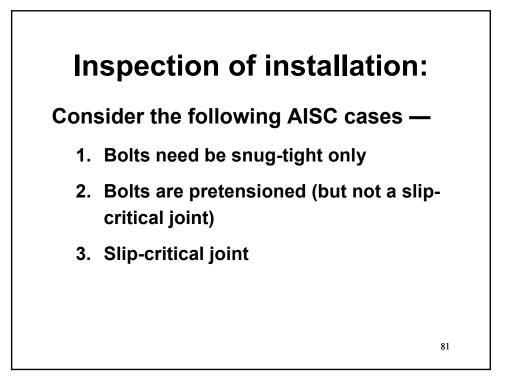


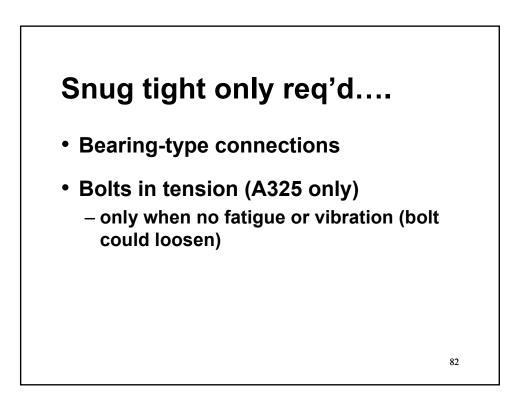






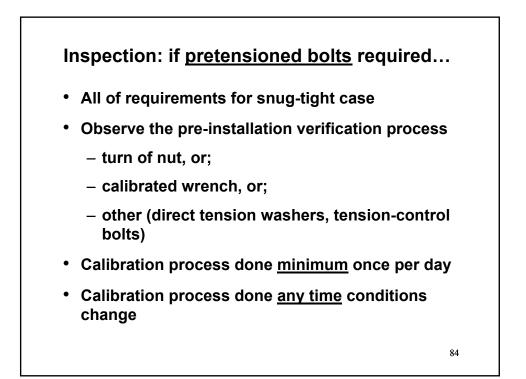


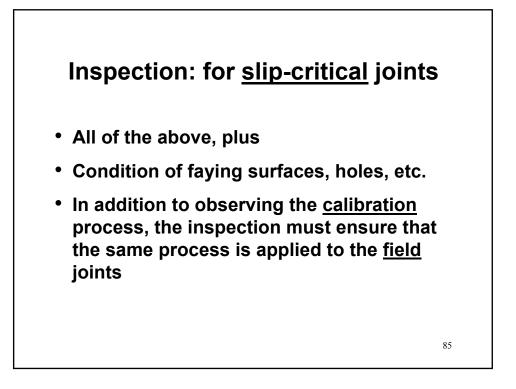


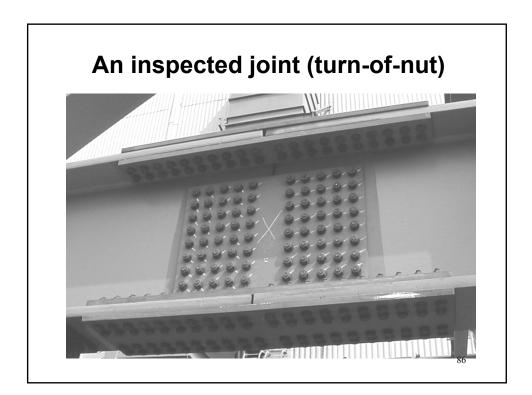


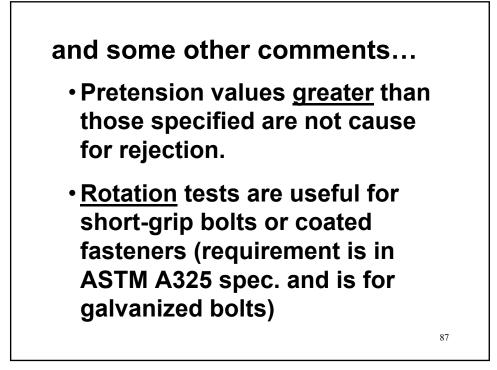
#### Inspection – <u>snug tight</u>

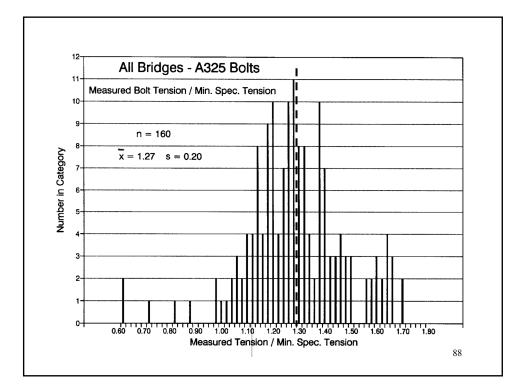
- Bolts, nuts, and washers (if any) must meet the requirements of the specifications
- Hole types (e.g., slotted, oversize) must meet specified requirements
- Contact surfaces are reasonably clean
- Parts are in close contact after bolts snugged
- All material within bolt grip must be steel









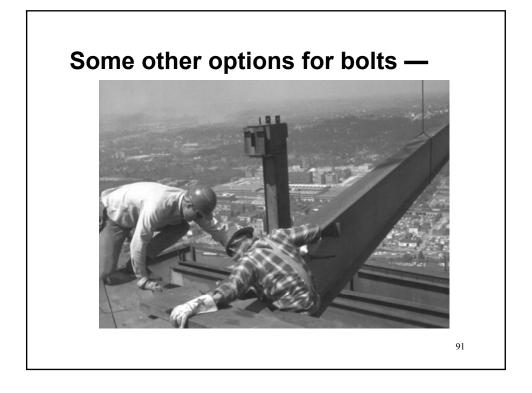


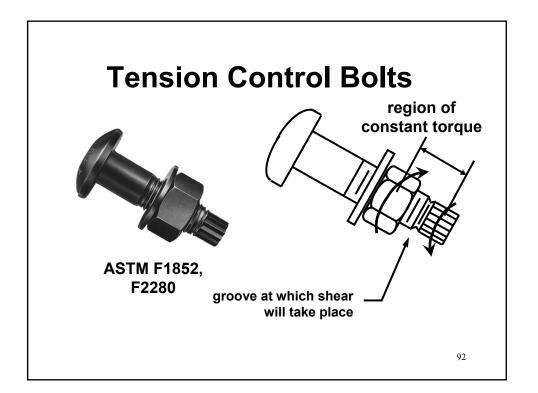
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# Actual pretensions, cont'd

- For A325 bolts, turn-of-nut:
  - Average <u>tensile strength</u> exceeds spec.
     min. tensile by about 1.18
  - Average <u>pretension force</u> is 80% of actual tensile
  - Result is that actual bolt tension is about 35% greater than specified bolt tension

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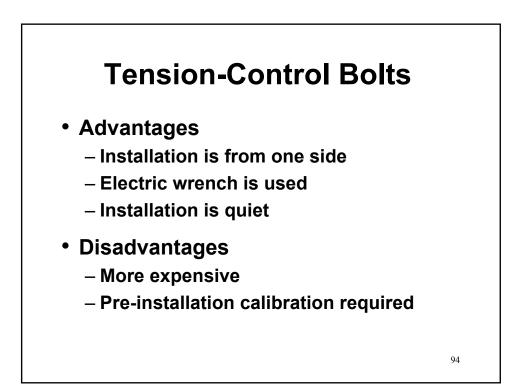




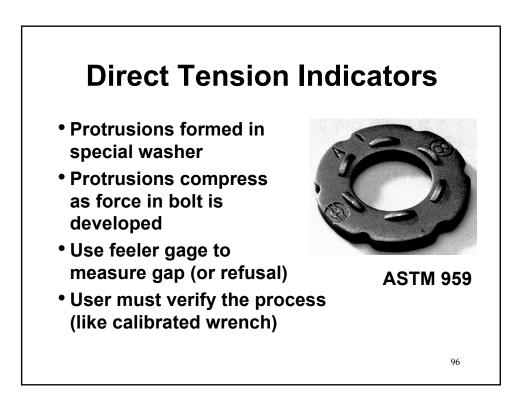
93

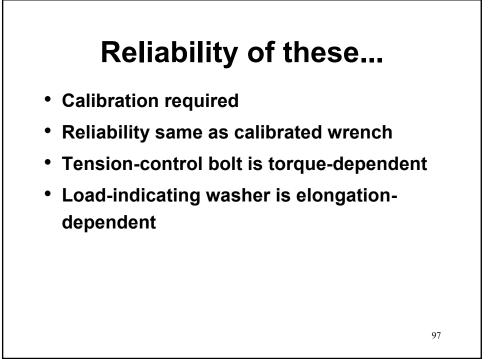
Tension control bolts....

- NOTE: evidence that tips have sheared off <u>is not</u> in itself evidence that desired pretension is present
- Consider limits:
  - Friction conditions are very high...
  - Friction conditions are very low...
- Hence, calibration is essential!

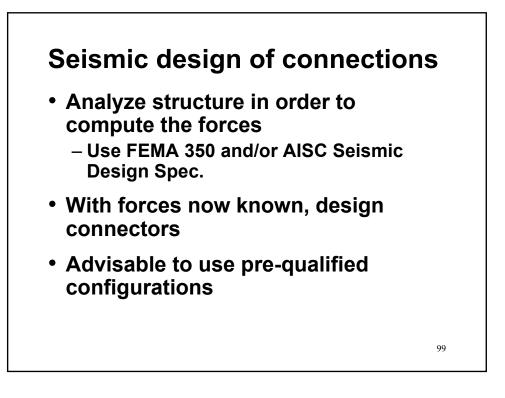


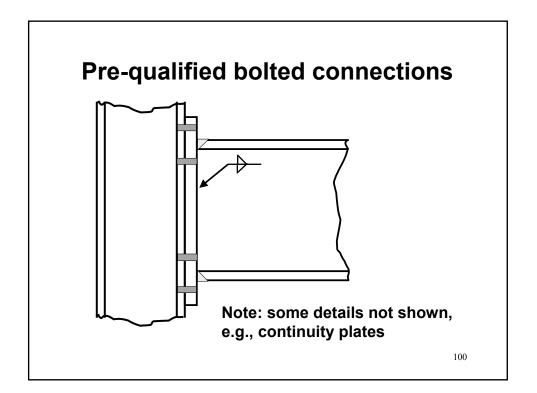


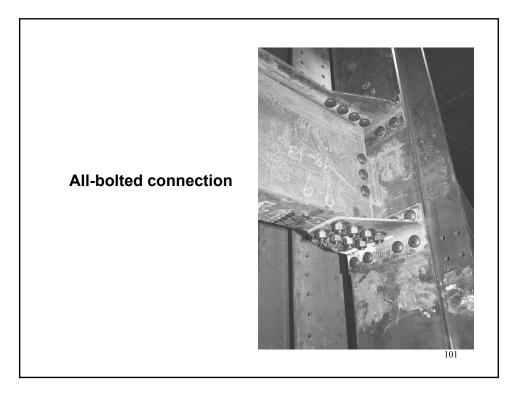


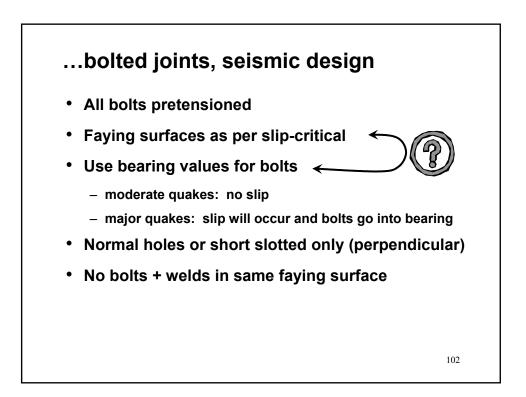


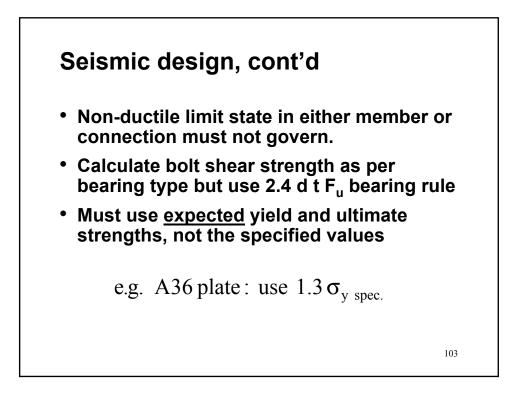


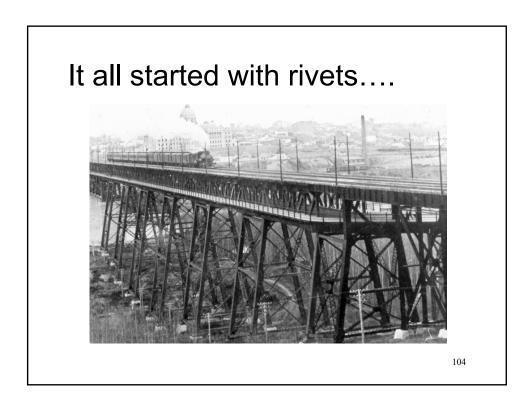


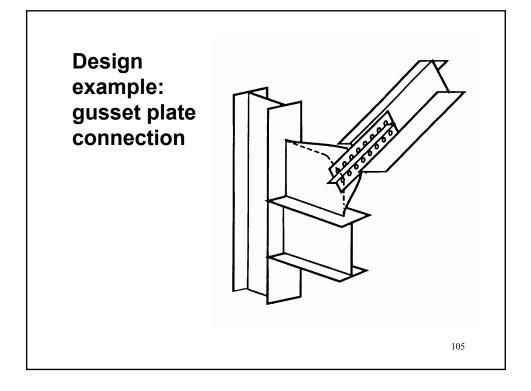


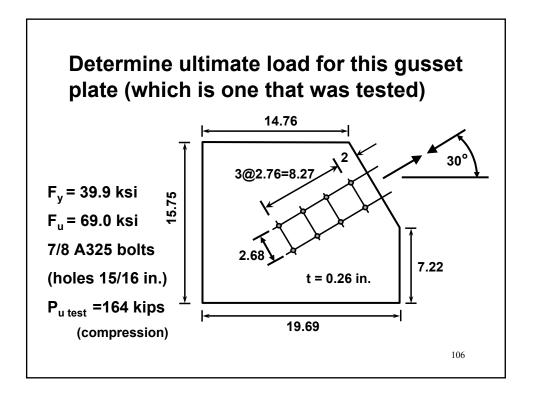


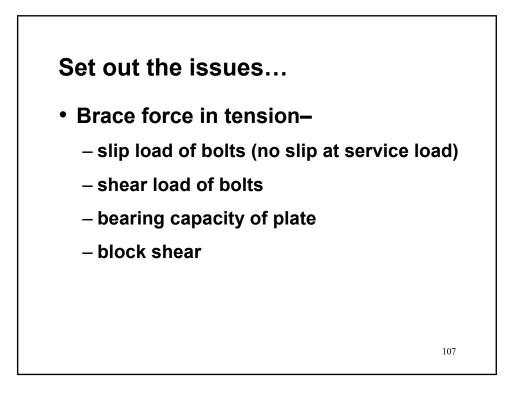


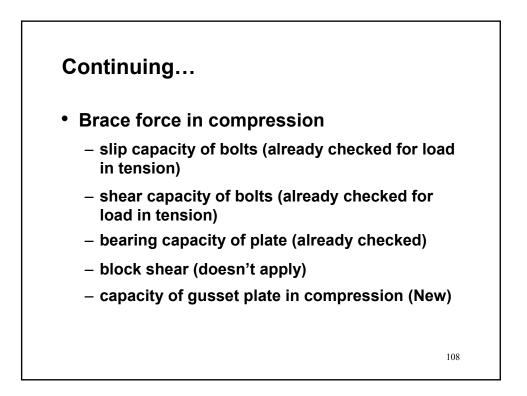








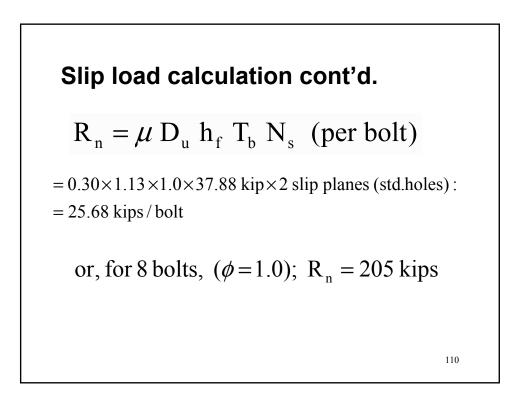




Slip load (calculate at factored load level)  

$$\begin{aligned} & R_n = \mu D_u h_f T_b n_s \text{ (per bolt)} \\ & \mu = 0.30 \text{ (clean mill scale)} h_f = 1.0 \text{ (no fills)} \text{)} \\ & A_b = \pi d^2 / 4 = 0.60 \text{ in}.^2 (7/8 \text{ in.dia.}) \\ & F_u = 120 \text{ ksi} (A325 \text{ bolts}) \\ & n_s = 2 \text{ slip planes} \end{aligned}$$

$$T_b = \text{spec.min.bolt pretension} = (0.75 \times A_b)(F_u)70\% \\ &= 0.75 \times 0.60 \text{ in}.^2 \times 120 \text{ ksi} \times 70\% = 37.88 \text{ kips} \end{aligned}$$



### Shear resistance of bolts

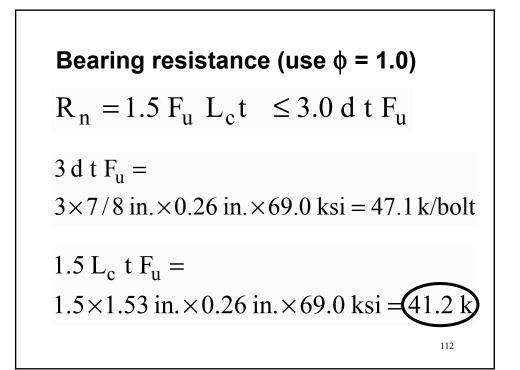
$$\phi R_n = \phi F_v A_b$$

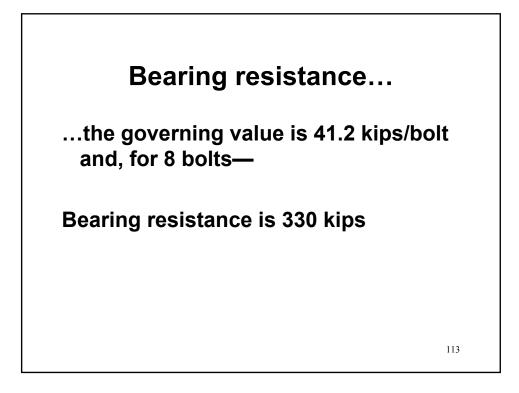
Use  $\emptyset = 1.0$  so that we can compare this load with the test load, assume threads in shear plane, no joint length effect

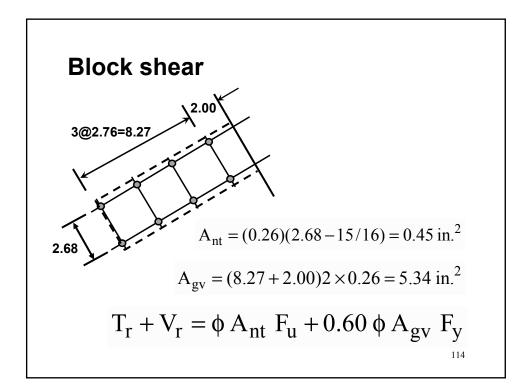
$$F_v = 90\% [0.62 \times 120 \text{ ksi}] = 68 \text{ ksi}$$

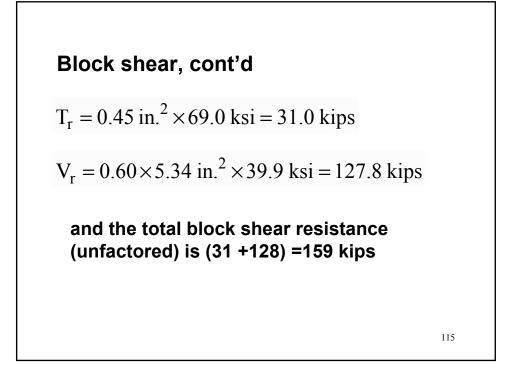
 $\phi R_n = 1.0 \times 68 \text{ ksi} \times 0.60 \text{ in.}^2 = 41.0 \text{ kips}$  (per bolt) or, for 8 bolts, 2 shear planes, threads in shear plane

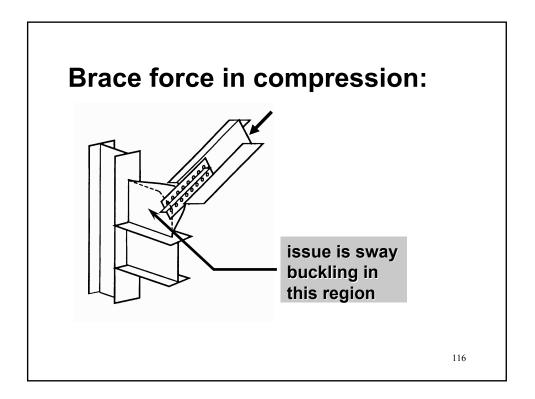
=  $(41.0 \times 8 \times 2)$ kips  $\times 0.80 = 525$  kips





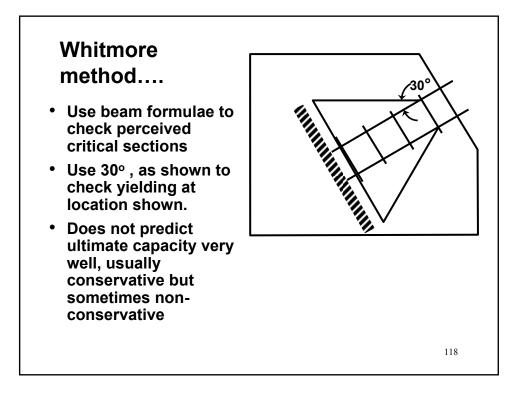


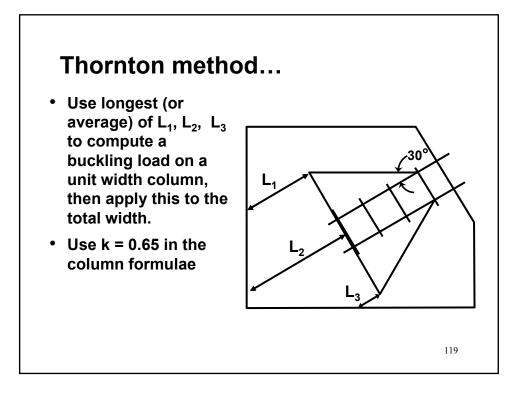


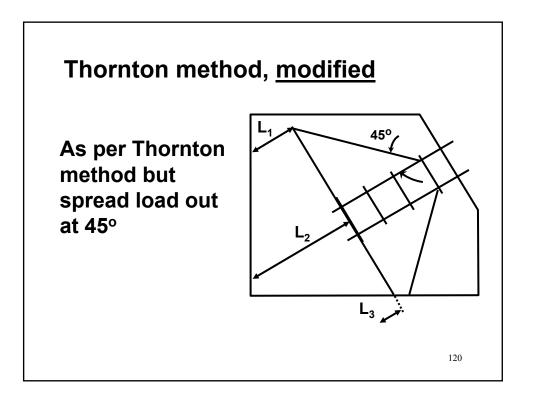


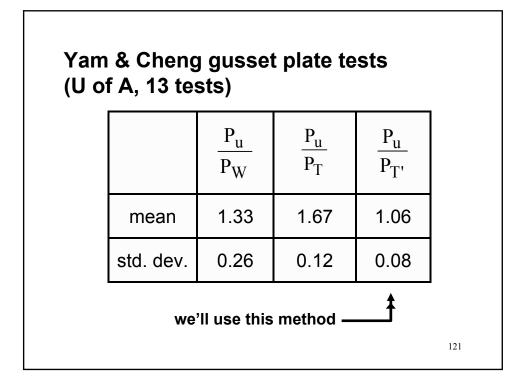
## Checking the buckling...

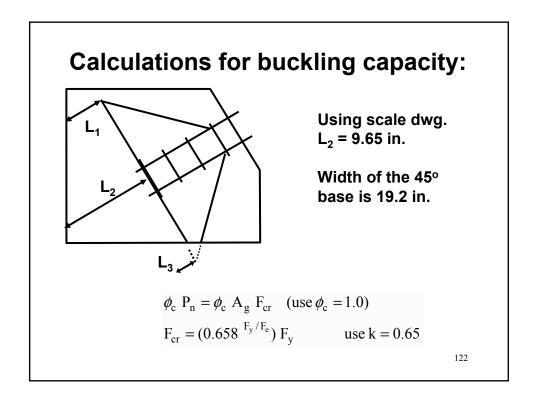
- Whitmore method (checks yield)
- Thornton method (checks buckling)
- Modified Thornton method (checks buckling)

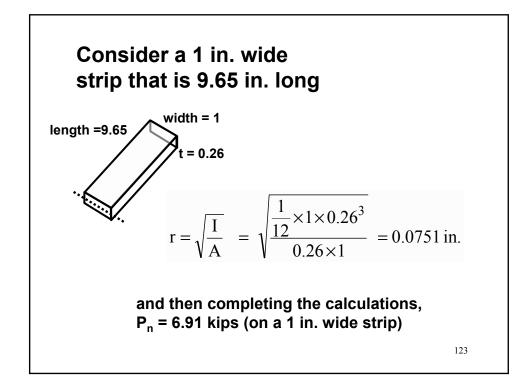


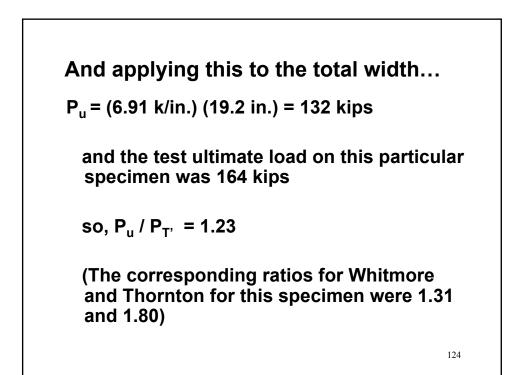






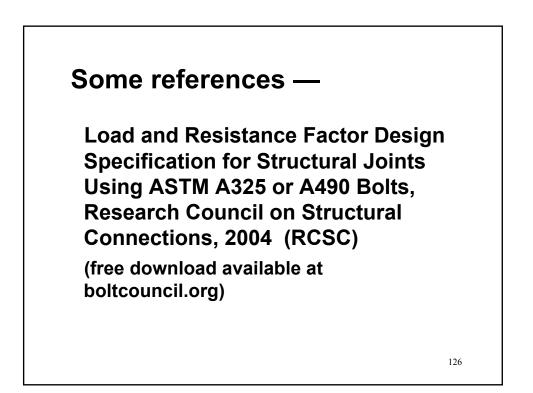






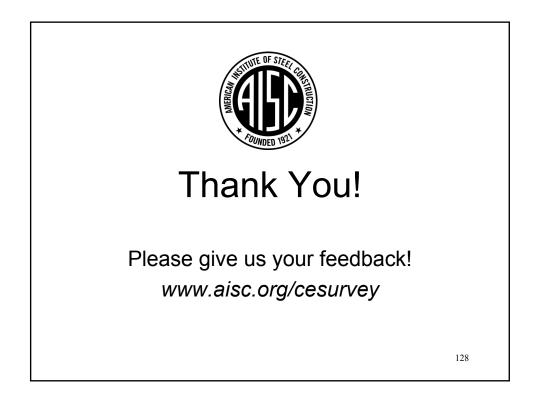
Γ

bolt shear	plate bearing	block shear	buckling	test load
1				
525	330	159	_	
_	_	_	132	164
	525 —	525 330 — —	525     330     159       -     -     -	



### References, cont'd.

- G.L. Kulak, J.W. Fisher, and J.A.H. Struik, *Guide to Design Criteria for Bolted and Riveted Joints*, Second Edition, John Wiley, New York, 1987 (free download at RCSC website)
- Bickford, John H., "An Introduction to the Design and Behavior of Bolted Joints," Second Edition, Marcel Dekker Inc., New York, 1990
- G.L. Kulak, A Bolting Primer for Structural Engineers, AISC Design Guide 17, Chicago, 2002
- Larry Kloiber and Larry Muir, "The 2010 AISC Specification: Changes in Design of Connections," Modern Steel Construction, Sept. 2010



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